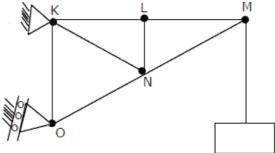
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Q.1-30 Carry One Mark Each

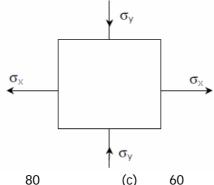
- If $x = a (\theta + \sin \theta)$ and $y = a (1 \cos \theta)$, then $\frac{dy}{dx}$ will be equal to 1.
- (b) $\cos\left(\frac{\theta}{2}\right)$ (c) $\tan\left(\frac{\theta}{2}\right)$
- The angle between two unit-magnitude coplanar vectors P(0.86, 0.500,0) and Q(0.259, 0.956, 0) will 2. be
 - 0° (a)
- (b) 30°
- 45° (c)
- The sum of the eigen values of the matrix given below is 3.
 - (a)
- (c)
- 4. The figure shows a pin-jointed plane truss loaded at the point M by hanging a mass of 100 kg. The member LN of the truss is subjected to a load of



(a) 0 Newton

- (b) 490 Newtons in compression
- (c) 981 Newtons in compression
- (d) 981 Newtons in tension
- 5. In terms of Poisson's ratio (v) the ratio of Young's Modulus (E) to Shear Modulus (G) of elastic materials is
 - 2(1 + v)(a)

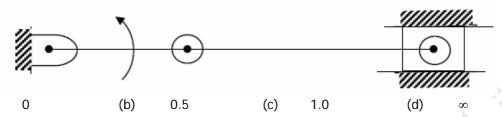
- 2(1 -v) (c) $\frac{1}{2}(1+v)$ (d) $\frac{1}{2}(1-v)$
- Two mating spur gears have 40 and 120 teeth respectively. The pinion rotates at 1200 rpm and 6. transmits a torque of 20 N.m. The torque transmitted by the gear is
 - (a) 6.6 Nm
- (b) 20 Nm
- 40 Nm
- (d) 60 Nm
- The figure shows the state of stress at a certain point in a stressed body. The magnitudes of normal stresses in the x and y direction are 100 MPa respectively. The radius of Mohr's stress circle representing this state of stress is



- (a) 120
- (b) 80
- (c)
- 40 (d)

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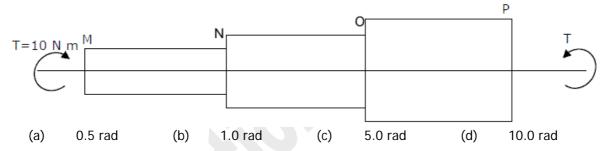
8. For a mechanism shown below, the mechanical advantage for the given configuration is



- 9. A vibrating machine is isolated from the floor using springs. If the ratio of excitation frequency of vibration of machine to the natural frequency of the isolation system is equal to 0.5, the transmissibility of ratio of isolation is
 - (a) $\frac{1}{2}$

(a)

- (b) $\frac{3}{4}$
- (c) $\frac{4}{3}$
- (d) 2
- 10. A torque of 10 Nm is transmitted through a stepped shaft as shown in figure. The torsional stiffnesses of individual sections of lengths MN, NO and OP are 20 Nm/rad, 30 Nm/rad and 60 Nm respectively. The angular deflection between the ends M and P of the shaft is



- 11. In terms of theoretical stress concentration factor (K_t) and fatigue stress concentration factor (K_f) , the notch sensitivity 'q' is expressed as
 - (a) $\frac{(k_f 1)}{(k_f 1)}$
- (b) $\frac{(k_f 1)}{(k_t + 1)}$
- (c) $\frac{(k_t 1)}{(k_f 1)}$
- (d) $\frac{(k_f + 1)}{(k_t 1)}$
- 12. The S-N curve for steel becomes asymptotic nearly at
 - (a) 10³ cycles
- (b) 10⁴ cycles
- c) 10⁶ cycles
- (d) 10⁹ cycles
- 13. In the window air conditioner, the expansion device used is
 - (a) capillary tube

- (b) thermostatic expansion valve
- (c) automatic expansion valve
- (d) float valve
- 14. During chemical dehumidification process of air
 - (a) dry bulb temperature and specific humidity decrease
 - (b) dry bulb temperature increases and specific humidity decreases
 - (c) dry bulb temperature decreases and specific humidity increases
 - (d) dry bulb temperature and specific humidity increase
- 15. At the time of starting, idling and low speed operation, the carburetor supplies a mixture which can be termed as
 - (a) lean

(b) slightly leaner than stoichiometric

(c) stoichimetric

(d) rich

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16.		mensional unstea an be written as	ady state	e heat transfer e	quation 1	for a sphere with	n heat ge	eneration at the rate
	(a)	$\frac{1}{r}\frac{\partial}{\partial r}\bigg(r\frac{\partial T}{\partial r}\bigg) + \frac{q}{k}$	$=\frac{1}{\alpha}\frac{\partial T}{\partial t}$		(b)	$\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial T}{\partial r} \right) +$	$\frac{q}{k} = \frac{1}{\alpha} \frac{d}{dx}$	∂T ∂t
	(c)	$\frac{\partial^2 T}{\partial r^2} + \frac{q}{k} = \frac{1}{\alpha} \frac{\partial}{\partial r^2}$	<u>∍T</u> ∋t		(d)	$\frac{\partial^2}{\partial r^2}(rT) + \frac{q}{k} = -\frac{q}{r^2}$	$\frac{1}{\alpha} \frac{\partial T}{\partial t}$	
17.	two pa	rallel plates. If th	ne top la ains a lin	te is moved with lear velocity prof	a veloci	ity of 0.5 m/s wh	nile the b	0.88) is held between cottom one is held in these plates; the
	(a)	0.651 × 10 ⁻³			(c)	6.51	(d)	0.651×10^3
18.		al formula is	_			w generation do		_
	(a)	CH CI F ₂	(b)	$C_2 CI_3 F_3$	(c)	C ₂ Cl ₂ F ₄	(d)	$C_2 H_2 F_4$
19.		flow is represent line passing thro	ugh a po	oint (1,2) is				tant. The equation of
	(a)	x - 2y = 0	(b)	2x + y = 0	(c)	2x - y = 0	(d)	x + 2y = 0
20.	the pro		ction of	200 kJ causes the ocess is	ne surroi			being 5000 kJ. During he change in internal
	(a)	-7000 kJ	(b)	-3000 kJ	(c)	+3000 kJ	(d)	+ 7000 kJ
21.		emperature limit	s of T _{min}			v	mum wo	ork output for the
	(a)	$\left(\frac{T_{max}}{T_{min}}\right)^{\frac{y}{2(y-1)}}$				$\left(\frac{T_{min}}{T_{max}}\right)^{\frac{y}{2(y-1)}}$		
	(c)	$\left(\frac{T_{max}}{T_{min}}\right)^{\frac{y-1}{y}}$			(d)	$\left(\frac{T_{min}}{T_{max}}\right)^{\frac{y-1}{y}}$		
22.	In an ir	nterchangeable a	nssembly mm. The (b)	y, shafts of size 2 e maximum poss 20 microns	25.000 ⁺⁰ ible clea (c)	n.040-0.01000 mm m rance in the asso 30 microns	nate with embly w (d)	n holes of ill be 60 microns
23.	During motion		a CNC _I	oart program blo	ock NO20) GO2 X45.0 Y25	.0 R5.0	the type of tool
	(a) (c)	circular Interpo linear Interpola		clockwise	(b) (d)	circular Interpo rapid feed	lation -	counterclockwise
24.	The me (a) (c)	echanism of mate Melting and Eva Erosion and Ca	aporatio	•	cess is (b) (d)	Melting and Co Cavitation and		tion
25.						current of 5000 of 0.2 second, he		ming effective erated during the

(d)

1000 Joules

5 Joule

(c)

process will be

0.2 Joule

(b)

1 Joule

(a)

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26.	In PERT (a) (c)	Fanalysis a critic maximum Float maximum Cost	al activi	ty has		(b) (d)		Float num Cost			
27.		roduct, the foreca onential smoothi									
	(a)	21	(b)	23		(c) 24			(d)	27	
28.	Produce P Q	tre two products Demand (Units) 100 400 Denomic order qua	Order Cost(F 50 50	Rs/orde	Holdin cr) Cost(F 4 1	ig Rs./unit/	/year)		io		
	(a)	1:1	(b)	1:2	•	(c)	1:4	oe iii tile tal	(d)	1:8	
29.	Misrun (a) (b) (c) (d)	is a casting defect very high pouring insufficient fluid absorption of gas improper alignm	ig tempe ity of thuses by	erature e molte the liqu	of the me en metal uid metal	etal					
30.	The per (a) (c)	rcentage of carbo 0.25 to 0.75 per 3 to 4 percent	cent			(b) (d)	1.25 8 to	to 1.75 per 10 percent	cent		
31.	The foll	owing data abou	t the flo	w of lie	quid was o	bserved	d in a	continuous	chemic	al pro	cess plant
		ate(litres/sec)	7.5 to	o 7.7	7.7 to 7.9		o 8.1	8.1 to 8.3		8.5	8.5 to 8.7
	Freque	ency ow rate of the lic	1 uid is		5	35		17	12		10
	(a)	8.00 litres/sec 8.16 litres/sec	julu is			(b) (d)		6 litres/sec litres/sec			
32.		pack of regular prds will be Kings,	if the f	irst car				random. W	hat is th	ne pro	bability that
	(a)	$\frac{1}{26}$ 26	(b) $\frac{1}{52}$			(c)	1 169		(d)	1 221	
33.	A delay	ed unit step func	tion is c	defined	as u(t–a)	$= \begin{cases} 0 \text{ for } t < \\ \\ 1 \text{ for } t \ge \end{cases}$	1 . Its 1	s Laplace tra	ansform	ı is	
	(a)	a.e ^{-as}	(b)	$\frac{e^{-as}}{s}$		(c)	$\frac{e^{as}}{s}$		(d)	$\frac{e^{as}}{a}$	

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34. The values of a function f(x) are tabulated below

Х	f(x)
0	1
1	2
2	1
3	10

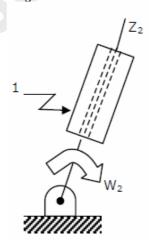
Using Newton's forward difference formula, the cubic polynomial that can be fitted to the above data,

- $2x^3 + 7x^2 6x + 2$ (a)
- $2x^{3} 7x^{2} + 6x 2$ $x^{3} 7x^{2} 6x + 1$ (b)
- (c)
- $2x^2 7x^2 + 6x + 1$
- The volume of an object expressed in spherical co-ordinates is given by $v = \sin \varnothing dr \varnothing d \varnothing d\theta h$ 35. The value of the integral is
 - (a)

- For which value of x will be matrix given below become singular? 36.

$$\begin{pmatrix}
8 & x & 0 \\
4 & 0 & 2 \\
12 & 6 & 0
\end{pmatrix}$$

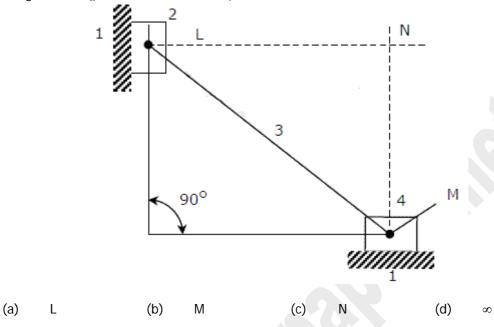
- (a)
- (b)
- (d) 12
- In the figure shown, the relative velocity of link 1 with respect to link 2 is 12 m/sec. Link 2 rotates at 37. a constant speed of 120 rpm. The magnitude of Coriolis component of acceleration of link 1 is



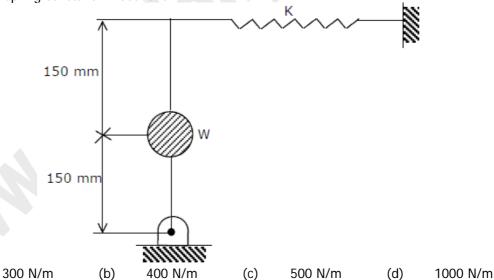
- 302 m/s^2 (a)
- 604 m/s² (b)
- 906 m/s² (c)
- 1208 m/s² (d)

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38. The figure below shows a planar mechanism with single degree of freedom. The instant center 24 for the given configuration is located at a position

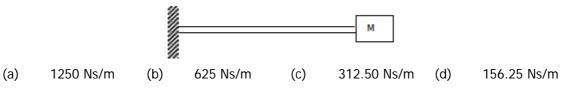


39. A uniform stiff rod of length 300 mm and having a weight of 300 N is pivoted at one end and connected to a spring at the other end. For keeping the rod vertical ins a stable position the minimum value of spring constant K needed is K



40. A mass M, of 20 kg is attached to the free end of a steel cantilever beam of length 1000 mm having a cross-section of 25×25 mm. Assume the mass of the cantilever to be negligible and 200 $E_{steel} = 200$ GPa. If the lateral vibration of this system is critically damped using a viscous damper, the damping constant of the damper is

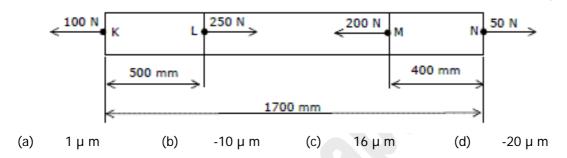
(a)



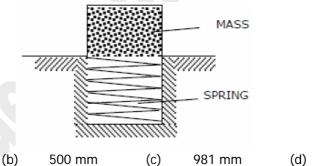
GATE MECHANICAL ENGINEERING 2004 (ME)

- 41. In a bolted joint two members are connected with an axial tightening force of 2200 N. if the bolt used has metric threads of 4 mm pitch, the torque required for achieving the tightening force is

- (a) 0.7 Nm
- (b) 1.0 Nm
- (c) 1.4 Nm
- (d) 2.8 Nm
- The figure below shows a steel rod of 25 mm² cross sectional area. It is loaded at four points, K, L, M 42. and N. Assume $E_{steel} = 200$ GPa. The total change in length of the rod due to loading is



43. An ejector mechanism consists of a helical compression spring having a spring constant of $K = 981 \times 10^3$ N/m. it is pre-compressed by 100 mm from its free state. If it is used to eject a mass of 100 kg held on it, the mass will move up through a distance of

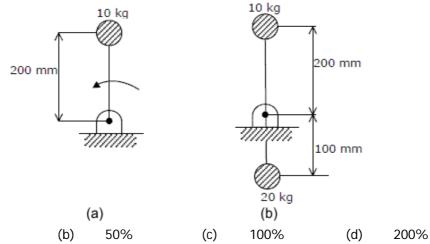


(a) 100 mm

(a)

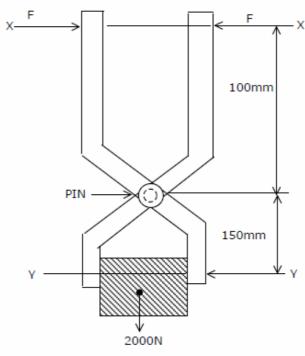
25%

- 1000 mm
- 44. A rigid body shown in the Fig.(a) has a mass of 10 kg. It rotates with a uniform angular velocity 'ω'. A balancing mass of 20 kg is attached as shown in Fig. (b). The percentage increase in mass moment of inertia as a result of this addition is 20 kg



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45. The figure shows a pair of pin-jointed gripper tongs holding an object weighing 2000 N. the coefficient of friction (µ) at the gripping surface is 0.1XX is the line of action of the input force and YY is the line of application of gripping force. If the pin-joint is assumed to be frictionless, the magnitude of force F required to hold the weight is



- (a) 1000 N
- (b) 2000 N
- (c) 2500 N
- (d) 5000 N
- 46. A solid circular shaft of 60 mm diameter transmits a torque of 1600 N.m. The value of maximum shear stress develop is
 - 37.72 MPa (a)
- (b) 47.72 MPa
- (c) 57.72 MPa
- (d) 67.72 MPa
- 47. For a fluid flow through a divergent pipe of length L having inlet and outlet radii and R₁ and R₂ respectively and a constant flow rate of Q, assuming the velocity to be axial and uniform at any cross section, the acceleration at the exit is
 - (a)
- (b) $\frac{2Q^2(R_1 R_2)}{\pi L R_2^3}$ (c) $\frac{2Q^2(R_1 R_2)}{\pi^2 L R_2^5}$ (d) $\frac{2Q^2(R_2 R_1)}{\pi^2 L R_2^5}$
- A closed cylinder having a radius R and height H is filled with oil density p. If the cylinder is rotated about its axis at an angular velocity of ω , the thrust at the bottom of the cylinder is
 - $\pi R^2 \rho aH$ (a)

 $\pi R^2 (\rho \omega^2 R^2 + \rho q H)$ (c)

- (b) $\pi R^2 \frac{p\omega^2 R^2}{4}$ (d) (b) $\pi R^2 \left(\frac{p\omega^2 R^2}{4} + pgH\right)$
- 49. For air flow over a flat plate, velocity (U) and boundary layer thickness (δ) can be expressed respectively, as $\frac{U}{U_a} = \frac{3}{2} \frac{y}{\delta} - \frac{1}{2} \left(\frac{y}{\delta}\right)^3$; $\delta = \frac{4.64x}{\sqrt{Re_x}}$

If the free stream velocity is 2m/s, and air has kinematic viscosity of 1.5×10^{-5} m² /s and density of 1.23 kg/m³, the wall shear stress at x = 1m, is

(a) $2.36 \times 10^{2} \text{ N/m}^{2}$

 $43.6 \times 10^{-3} \text{ N/m}^2$ (b)

 $4.36 \times 10^{-3} \text{ N/m}^2$ (c)

 $2.18 \times 10^{-3} \text{ N/m}^2$ (d)

									_
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50.	locatio averag	n of the pump th	rough a r in the إ	pipe of diamete pipe is 2 m/s. if	er 0.2 m it is mair	having Darcy' ntain a consta	's friction fa nt head of	km away from the actor of 0.01. The 5 m in the tank, is is	
51.	perforr		energy a	absorbed from lo	ow tempe	erature reserv	oir by the	a co-efficient of refrigerator for each	
52.	of a flu rejects	uid at an overall	efficiency f the he	y of 50%. The fl	uid heat	ed to 350 K is	s used to ru	to the internal energy in a heat engine which im area of the solar 79.36 m ²	
53.		essure gauges G .00 bar. The valu				show pressur	res of P _{G1} =	5.00 bar and	
	(a)	1.01 bar	(b)	2.01 bar	(c)	5.00 bar	(d)	7.01 bar	

- 54. A steel billet of 2000 kg mass is to be cooled from 1250 K to 450 K. The heat released during this process is to be used as a source of energy. The ambient temperature is 303 K and specific heat of steel is 0.5 kJ/kg K. the available energy of this billet is
 - (a) 1.01 bar
- (b) 2.01 bar
- (c) 5.00 bar
- (d) 7.01 bar
- 55. A stainless steel tub ($k_8 = 19 \text{ W/mK}$) of 2 cm ID and 5 cm OD is insulated with 3 cm thick asbestos ($k_a = 0.2 \text{ W/mK}$). If the temperature difference between the innermost and outermost surfaces is 600° C, the heat transfer rate per unit length is
 - (a) 0.94 W/m
- (b) 9.44 W/m
- (c) 944.72 W/m
- (d) 9447.21 W/m
- A spherical thermocouple junction of diameter 0.706 mm is to be used for the measurement of temperature of a gas stream. The convective heat transfer co-efficient on the bead surface is 400 W/m² K. Thermophysical properties of thermocouple material are k = 20 W/mK, C = 400 J/kg K and p = 8500 / kg m³ . If the thermocouple initially at 30° C is placed in a hot stream of 300° C, the time taken by the bead to reach 298° C, is
 - (a) 2.35 s
- (b) 4.9 s
- (c) 14.7 s

 70.7 m^2

- (d) 29.4 s
- 57. In a condenser, water enters a 30° C and flows at the rate 1500 kg/hr. The condensing steam is at a temperature of 120° C and cooling water leaves the condenser at 80° C. Specific heat of water is 4.187 kJ/kg K. If the overall heat transfer coefficient is 2000 W/m² K, the heat transfer area is 2
 - (a) 0.707 m^2
- (b) 7.07 m^2
- (c)
- (d) 141.4 m^2

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58. During a Morse test on a 4 cylinder engine, the following measurements of brake power were taken at constant speed.

All cylinders firing	3037 kW
Number 1 cylinder not firing	2102 kW
Number 2 cylinder not firing	2102 kW
Number 3 cylinder not firing	2100 kW
Number 4 cylinder not firing	2098 kW

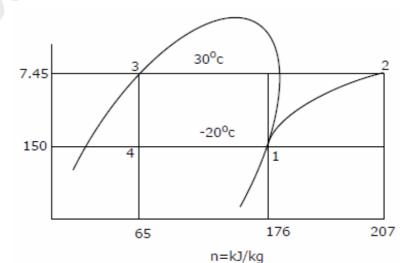
The mechanical efficiency of the engine is

- (a) 91.53%
- (b)
- 85.07%
- (c) 81.07%
- (d) 61.22
- 59. An engine working on air standard Otto cycle has a cylinder diameter of 10 cm and stroke length of 15 cm. The ratio of specific heats for air is 1.4. If the clearance volume is 196.3 cc and the heat supplied per kg of air per cycle is 1800 kJ/kg, the work output per cycle per kg of air is
 - (a) 879.1 kJ
- (b) 890.2 kJ
- (c) 895.3 kJ
- (d) 973.5 k.
- 60. At a hydro electric power plant site, available head and flow rate are 24.5 m and 10.1 m³/s respectively. If the turbine to be installed is required to run at 4.0 revolution per second (rps) with an overall efficiency of 90%, the suitable type of turbine for this site is
 - (a) Francis
- b) Kaplan
- c) Pelton
- (d) Propeller
- 61. Dew point temperature of air at one atmospheric pressure (1.013 bar) is 18° C. The air dry bulb temperature is 30° C. The saturation pressure of water at 18° C and 30° C are 0.02062 bar and 0.04241 bar respectively. The specific heat of air and water vapour respectively are 1.005 and 1.88 kJ/kg K and the latent heat of vaporization of water at 0° C is 2500 kJ/kg. The specific humidity (kg/kg of dry air) and enthalpy (kJ/kg of dry air) of this moist air respectively, are
 - (a) 0.01051, 52.64

(b) 0.01291, 63.15

(c) 0.01481, 78.60

- (d) 0.01532, 81.40
- 62. A R-12 refrigerant reciprocating compressor operates between the condensing temperature of 30° C and evaporator temperature of -20° C. The clearance volume ratio of the compressor is 0.03. Specific heat ratio of the vapour is 1.15 and the specific volume at the suction is 0.1089 m³ /kg. Other properties at various states are given in the figure. To realize 2 Tons of refrigeration, the actual volume displacement rate considering the effect of clearance is



(a) $6.35 \times 10^{-3} \text{ m}^3 \text{ /s}$

- (b) 63
- $63.5 \times 10^{-3} \text{ m}^3 \text{/s}$

(c) $635 \times 10^{-3} \text{ m}^3 / \text{s}$

(d) $4.88 \times 10^{-3} \text{ m}^3 /\text{s}$

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							+0.050		
63.	taken as		le tolera						tolerances can be of GO and NO-GO
		20.010 mm and	•) mm		(b)	20.014 mm and 20.04		mm
		20.006 mm and				(d)	20.014 mm a	and 20.054	mm
	. ,					` ,			
64.							•	npared for	the production of a
	compon	ent. Following o	data refe				Automatic		
				Tool	Standard Machine		Machine Tool		
		Setup time		30 min			2 hours		
		Setup time		30 111111			2 110013		
		Machining tir per piece	ne	22 min			5 min		
		Machine rate		Rs.200 p	er hour		Rs.800 per hou	ır	
	The brea	akeven product	ion batc	h size abo	ove which	h the	automatic mach	 nine tool w	rill be economical to
	use, will								
	(a)	4	(b)	5		(c)	24	(d)	225
65.	material force du	is 400 N/mm ² Iring the operat	and pen ion will l	etration is be	s 40%. S	Shear	provided on the	e punch is	ear strength of the 2 mm. The blanking
	(a)	22.6 kN	(b)	37.7 kN		(c)	61.6 kN	(d)	94.3 kN
66.	speed is the time		0.2 mm	/rev and	drill poir			ıming drill	ckness. Drill spindle overtravel of 2 mm, 110 seconds
67.	In a 2-D	CAD nackage	clockwie	so circular	arc of r	radius	5 specified fro	om P./15	10) to P ₂ (10,15) will
07.		center at	CIOCKVVI	se circular	arc or i	aulus	, o, specified fre)III I 1(13,	10) to 1 2(10, 13) Will
	(a)	(10, 10)	(b)	(15, 10))	(c)	(15, 15)	(d)	(10, 15)
68.	pattern						est in sand moul ern to that of th 1.01		age allowance for vill be 1.03
69.	Using M	Cutting speed Depth of cut Tool rake angle Chip thickness Cutting force Thrust force	÷	: : : :	40 m/m 0.3 mm + 5° 1.5 mm 900 N 450 N	in	ng data were ob e machining wil 45°		63.4°
70.		ing process, she it rotates at 10						kness. Roll	is of diameter 600
	(a)	5 mm	(h)	39 mm		(c)	78 mm	(d)	120 mm

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71.	In a machining operation,	doubling the cutting	speed reduces t	the tool life to $\frac{1_{tl}}{8}$	h of the original	value.
				U	,	

The exponent n in Taylor's tool life equation $VT^n = C$, is

(a)
$$\frac{1}{8}$$

(d)
$$\frac{1}{2}$$

72. A soldering operation was work-sampled over two days (16 hours) during which an employee soldered 108 joints. Actual working time was 90% of the total time and the performance rating was estimated to be 120 percent. If the contract provides allowance of 20 percent of the total time available, the standard time for the operation would be

(a) 8 min.

(b)

8.9 min

(c)

10 min.

d) 12 min.

73. An electronic equipment manufacturer has decided to add a component sub-assembly operation that can produce 80 units during a regular 8-hour shift. This operation consists of three activities as below

Activity

Standard time (min.)

M. Mechanical assembly
E. Electric wiring
T. Test

16 3

12

For line balancing the number of work stations required for the activities M, E and T would respectively be

(a) 2, 3, 1

(b)

3, 2,1

(c) 2, 4, 2

(d) 2, 1, 3

74. A maintenance service facility has Poisson arrival rates, negative exponential service time and operates on a 'first come first served' queue discipline. Breakdowns occur on an average of 3 per day with a range of zero to eight. The maintenance crew can service an average of 6 machines per day with a range of zero to seven.

The mean waiting time for an item to be serviced would be

(a)
$$\frac{1}{6}$$
 day

(b)
$$\frac{1}{3}$$
 day

75. A company has an annual demand of 1000 units, ordering cost of Rs.100/order and carrying cost of Rs.100/unit -year. If the stock-out costs are estimated to be nearly Rs.400 each time the company runs out-of-stock, the safety stock justified by the carrying cost will be

(a)

(b)

20 20 20

(c)

(d) 100

76. A company produces two types of toys: P and Q. Production time of Q is twice that of P and the company has a maximum of 2000 time units per day. The supply of raw material is just sufficient to produce 1500 toys (of any type) per day. Toy type Q requires an electric switch which is available @ 600 pieces per day only. The company makes a profit of Rs.3 and Rs.5 on type P and Q respectively. For maximization of profits, the daily production quantities of P and Q toys should respectively be (a) 100, 500 (b) 500, 100 (c) 800, 600 (d) 1000, 1000

77. Match the following

Type of Mechanism

- P. Scott œ Russel mechanism
- Q. Geneva mechanism
- R. Off-set slider-crank mechanism
- S. Scotch Yoke mechanism

P-2 Q-3 R-1 S-4 (a) (b) P-3 Q-2 S-1 (c) S-3 P-4 Q-1 R-2 (d) P-4 Q-3 R-1 S-2

Motion achieved

40

- 1. Intermittent motion
- 2. Quick return motion
- 3. Simple harmonic motion
- 4. Straight line motion

GATE MECHANICAL ENGINEERING 2004 (ME)

78.	Match	the	following.
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Type o	of gear	S	Arrangement of shafts								
P. Bevel gears			1. Non-pai	Non-parallel off-set shafts							
Q. Worm gears	5		2. Non-pai	2. Non-parallel intersecting shafts							
R. Herringbone	egears		3. Non-pai	3. Non-parallel, non-intersecting shafts							
S. Hypoid gear	S		4. Parallel	4. Parallel shafts							
(a) P-4	Q-2	R-1	S-3	(b)	P-2	Q-3	R-4	S-1			
(c) P-3	Q-2	R-1	S-4	(d)	P-1	Q-3	R-4	S-2			

79. Match the following with respect to spatial mechanisms.

			J	•	•					
	P. Revo Q. Cyli R. Sph	olute ndrical	of Join	t	Degree of of 1. Three 2. Five 3. Four 4.Two 5.Zero	constrain	t			
	(a)	P-4	Q-3	R-3	3.2610	(b)	P-5	Q-4	R-3	
	(c)	P-2	Q-3	R-1		(d)	P-4	Q-5	R-3	
80.	P. Reci Q. Axia R. Micr	al flow p ohydel	ng pump bump		 Plant with Plant with Positive d Draft tube High flow Centrifug 	power ou isplaceme e rate, low	itput bet nt pressur	tween 10		MW
	(a)	P-3	Q-5	R-6	S-2	(b)	P-3	Q-5	R-2	S-6
	(c)	P-3	Q-5	R-1	S-6	(d)	P-4	Q-5	R-1	S-6
81.	P. Pitch	n and A	i re to b ongle erro	e inspectors of sci	rew thread		o Collim	ument ator erferome	ter	
	Z. Hat	11033 011	or or a	Juliace L	nuto	2. Op	ioui iiit	,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	101	

P. Pitch and	Angle err	ors of so	rew thread	1. Auto Collimator						
Q. Flatness	error of a	surface	plate	2. Optical Interferometer						
R. Alignmen	t error of	a machii	ne slideway	3. Dividing Head and Dial Gauge						
S. Profile of	a cam			4. Spirit Level						
				5. Sine bar						
				Tool maker's Microscope						
(a) P-6	Q-2	R-4	S-6	(b)	P-5	Q-2	R-1	S-6		
(c) P-6	Q-4	R-1	S-3	(d)	P-1	Q-4	R-4	S-2		

Match	the foll	owing								
		Produ	uct							
P. Mo	lded lug	gage			1. Inj	ection m	nolding			
	_	containe	rs for lic	quid	2. Ho	t rolling	· ·			
R. Lo	ng struct	tural sha	pes		3. Im	3. Impact extrusion				
S. Co	llapsible	tubes			4. Transfer molding					
	-				5. Blo	w moldi	ng			
					6. Co	ining				
(a)	P-1	Q-4	R-6	S-3	(b)	P-4	Q-5	R-2	S-3	
(c)	P-1	Q-5	R-3	S-2	(d)	P-5	Q-1	R-2	S-2	
					` '					

GATE MECHANICAL ENGINEERING 2004 (ME)

83. Typical machining operations are to be performed on hand-to-machine materials by using the processes listed below. Choose the best set of Operation-process combinations

Operation

- P. Deburring (internal surface)
- Q. Die sinking
- R. Fine hole drilling in thin sheets
- S. Tool sharpening

Process

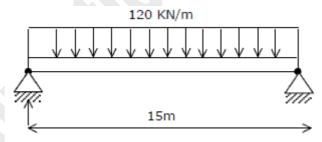
- 1. Plasma Arc Machining
- 2. Abrasive Flow Machining
- 3. Electric Discharge Machining
- 4. Ultrasonic Machining
- 5. Laser beam Machining
- 6. Electrochemical Grinding
- (a) P-1 0-5R-3 S-4
- (c) P-5 Q-1 R-2 S-6
- (b) P-1
- Q-4 R-1 (d) P-2 Q-3 R-5
- 84. From the lists given below, choose the most appropriate set of heat treatment process and the corresponding process characteristics

Process

Characteristics

- P. Tempering
- R. Martempering
- Q. Austempering
- 1. Austenite is converted into bainite
- 2. Austenite is converted into martensite
- 3. Cementite is converted into globular structure
- 4. Both hardness and brittleness are reduced
- 5. Carbon is absorbed into the metal
- (a) P-3 Q-1 R-5 (c) P-4 Q-1 R-2
- P-4 (b) Q-3 R-2 Q-5 (d) P-1 R-4

Data for Q.85-86 are given below. Solve the problems and choose correct answers. A steel beam of breadth 120 mm and height 750 mm is loaded as shown in the figure. Assume $E_{steel} = 200GPa$



- 85. The beam is subjected to a maximum bending moment of
 - 3375 kNm
- (b) 4750 kNm
- 6750 kNm
- 8750 kNm (d)

S-2

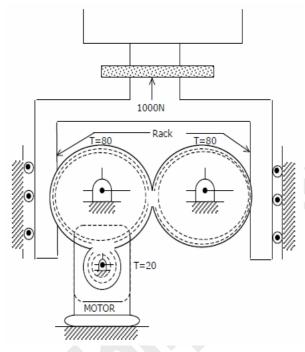
S-6

- 86. The value of maximum deflection of the beam is
 - (a) 93.75 mm
- (b) 83.75 mm
- (c) 73.75 mm
- (d) 63.75 mm

GATE MECHANICAL ENGINEERING 2004 (ME)

Data for Q.87-88 are given below. Solve the problems and choose correct answers.

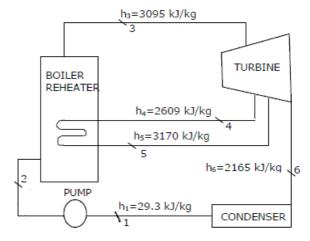
A compacting machine shown in the figure below is used to create a desired thrust force by using a rack and pinion arrangement. The input gear is mounted on the motor shaft. The gears have involutes teeth of 2 mm module.



- 87. If the drive efficiency is 80%, the torque required on the input shaft to create 1000 N output thrust is
 - (a) 20 Nm
- (b) 25 Nm
- (c) 32 Nm
- (d) 50 Nm
- 88. If the pressure angle of the rack is 20°, the force acting along the line of action between the rack and the gear teeth is
 - (a) 250 N
- (b) 342 N
- (c) 532 N
- (d) 600 N

Data for Q. 89 and 90 are given below. Solve the problem and choose correct answers.

Consider a steam power plant using a reheat cycle as shown. Steam leaves the boiler and enters the turbine at 4 MPa, 350° C ($h_3 = 3095 \text{ kJ/kg}$). After expansion in the turbine to 400 kPa ($h_4 = 2609 \text{ kJ/kg}$), the steam is reheated to 350° C ($h_5 = 3170 \text{ kJ/kg}$), and then expanded in a low pressure turbine 10 kPa ($h_6 = 2165 \text{ kJ/kg}$). The specific volume of liquid handled by the pump can be assumed to be



- 89. The thermal efficiency of the plant neglecting pump work is
 - (a) 15.8%
- (b) 41.1%
- (c) 48.5%
- (d) 58.6%

- 90. The enthalpy at the pump discharge (h_2) is
 - (a) 0.33 kJ/kg
- (b) 3.33 kJ/kg
- (c) 4.0 kJ/kg
- (d) 33.3 kJ/kg